

# On the near-wall characteristics of acceleration in turbulence

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The behaviour of fluid-particle acceleration in near-wall turbulent flows is investigated in numerically simulated turbulent channel flows at low to moderate Reynolds numbers,  $Re_\tau = 180 \sim 600$ . The acceleration is decomposed into pressure-gradient (irrotational) and viscous contributions (solenoidal acceleration) and the statistics of each component are analysed. In near-wall turbulent flows, the probability density function of acceleration is strongly dependent on the distance from the wall. Unexpectedly, the intermittency of acceleration is strongest in the viscous sublayer, where the acceleration flatness factor of  $O(100)$  is observed. It is shown that the centripetal acceleration around coherent vortical structures is an important source of the acceleration intermittency. We found sheet-like structures of strong solenoidal accelerations near the wall, which are associated with the background shear modified by the interaction between a streamwise vortex and the wall. We found that the acceleration Kolmogorov constant is a linear function of  $y^+$  in the log layer. The Reynolds number dependence of the acceleration statistics is investigated.

**Key words:** intermittency, turbulent boundary layers, turbulence simulation

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## 1. Introduction

Lagrangian acceleration is one of the fundamental variables in the Navier–Stokes equations and of great importance in understanding Lagrangian nature of turbulent flows. The study on acceleration is relevant not only to theoretical developments (Hill 2002; Biferale *et al.* 2004; Yeung *et al.* 2007) but also to many engineering applications, such as the prediction of near-surface dispersion of airborne pollutant (Sawford 1991; Pope 1994; Reynolds 1999) and particle deposition onto a surface (Brooke *et al.* 1992; Setyawati *et al.* 2002). Because of the Lagrangian nature, measurement of acceleration is a challenging task and only quite recently have direct measurements of acceleration become possible using optical imaging (La Porta *et al.* 2001). Due to developments of experimental techniques and high-performance computing, fluid-particle acceleration has been studied extensively over the last decade (see Toschi & Bodenschatz 2009 for a review). Recently, acceleration of finite-sized and/or inertial particles has begun to attract considerable attention (Qureshi *et al.* 2007; Volk *et al.* 2008; Calzavarnini *et al.* 2009; Yeo *et al.* 2010). Now, there is a large volume of literature on the behaviour

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of acceleration in isotropic turbulence and our understanding of acceleration in isotropic turbulence is much more advanced than that in inhomogeneous turbulent flows, such as turbulent shear layers. Still, there are only a limited number of studies on the behaviour of acceleration in near-wall turbulence (Lee, Yeo & Choi 2004; Gerashchenko *et al.* 2008; Yeo, Kim & Lee 2009).

Monin & Yaglom (1975) suggested that the Lagrangian acceleration is largely determined by the smallest scale motion, smaller than the Kolmogorov scale, and pressure forces mainly contribute to the acceleration at high Reynolds numbers. Hill & Thoroddsen (1997) estimated the two-point correlation of acceleration from the third- and fourth-order velocity structure functions to show that the viscous-force contribution to acceleration is negligible compared to that of the pressure force. Yeung (1997) found in the simulation of homogeneous shear flow that the one- and two-particle correlations of acceleration exhibit an approximately isotropic behaviour, which is consistent with the local isotropy.

According to the classical Kolmogorov similarity theory, the acceleration Kolmogorov constant, acceleration variance scaled by  $\epsilon$  and  $\nu$ , is a universal constant at sufficiently high Reynolds number. However, it is still not clear whether the acceleration variance exhibits the universal behaviour at high Reynolds number. Yeung & Pope (1989) found in the simulation of isotropic turbulence that the acceleration Kolmogorov constant is not universal and rather increases as  $Re_\lambda^{1/2}$ , where  $Re_\lambda$  denotes the Taylor-scale Reynolds number. Vedula & Yeung (1999) suggested that the Reynolds number dependence comes from the non-universal behaviour of pressure-gradient statistics. Gotoh & Rogallo (1999) proposed a theoretical model which assumes that the small-scale pressure field is driven by coherent structures to explain the Reynolds number dependence of acceleration statistics. La Porta *et al.* (2001) showed that acceleration is highly intermittent in their direct measurement of the Lagrangian acceleration in quasi-isotropic turbulence. They claimed that the Kolmogorov scaling is attained at high  $Re_\lambda$ . Yeung *et al.* (2006) performed direct numerical simulations (DNS) of isotropic turbulence for  $Re_\lambda = 40 \sim 680$  and suggested a fitting curve for the acceleration Kolmogorov constant. Recently, Gulitski *et al.* (2007) measured acceleration variances for  $Re_\lambda$  up to  $10^4$ . They showed that the acceleration Kolmogorov constant depends on  $Re_\lambda$  even at  $Re_\lambda \sim 10^4$  and the behaviour of the acceleration Kolmogorov constant is well predicted with the fitting curve by Yeung *et al.* (2006).

The Navier–Stokes equations relate the Lagrangian acceleration to the Eulerian variables (Monin & Yaglom 1975). The Lagrangian acceleration ( $\mathbf{a}$ ) is composed of two contributions from the pressure-gradient and the viscous forces. Conventionally, the pressure-gradient force is called potential or irrotational acceleration ( $\mathbf{a}^I$ ) and the viscous force is solenoidal acceleration ( $\mathbf{a}^S$ ), owing to their inherent characteristics (Monin & Yaglom 1975; Vedula & Yeung 1999). It has been discussed in isotropic turbulence that  $\mathbf{a}^I$  mainly contributes to  $\mathbf{a}$  and the correlation between  $\mathbf{a}$  and  $\mathbf{a}^S$  becomes negligible as the Reynolds number increases (Hill & Thoroddsen 1997; Tsinober, Vedula & Yeung 2001). In near-wall turbulence, however, the turbulent field is highly anisotropic and the viscous effects become dominant in the vicinity of the wall. Therefore, the behaviour of acceleration may be different from that in isotropic turbulence.

The main purpose of this study is to investigate the behaviour of acceleration in near-wall turbulent flow. Particularly, we focus on the connection between near-wall coherent structures and acceleration. We performed DNS of turbulent channel flow at low to moderate Reynolds number. We report various acceleration statistics which

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$Re_\tau$	$N_x \times N_y \times N_z$	$\Delta x^+$	$\Delta z^+$	$\Delta y_{min}^+$
180	$192 \times 129 \times 192$	11.8	3.93	0.05
408	$192 \times 193 \times 192$	13.1	4.36	0.05
600	$320 \times 257 \times 320$	11.8	3.93	0.05

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TABLE 1. Simulation parameters.  $N_x \times N_y \times N_z$  are the number of modes and  $\Delta x^+$ ,  $\Delta y^+$  and  $\Delta z^+$  are the numerical grid spacings normalized by the wall units in the streamwise, wall-normal and spanwise directions, respectively.

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have not been presented for near-wall turbulence. The different behaviours of  $\mathbf{a}$ ,  $\mathbf{a}^I$  and  $\mathbf{a}^S$  around the coherent structures are illustrated.

The remainder of this paper is organized as follows: numerical methods are briefly described in §2; acceleration statistics are analysed in §3; the behaviour of acceleration near a coherent structure is presented in §4; finally, the conclusions are provided in §5.

## 2. Numerical approach

The Navier–Stokes equations for incompressible flow are

$$\frac{\partial u_i}{\partial x_i} = 0, \quad (2.1)$$

$$\frac{Du_i}{Dt} = -\frac{\partial p}{\partial x_i} + \frac{1}{Re_\tau} \nabla^2 u_i, \quad (2.2)$$

where  $x_1, x_2$  and  $x_3$  denote the streamwise ( $x$ ), wall-normal ( $y$ ) and spanwise directions ( $z$ ), respectively. Here, the  $u_i$  are the corresponding velocity components ( $u, v, w$ ),  $p$  is pressure and  $Re_\tau$  is the Reynolds number based on the channel half-width  $\delta$  and the wall-shear velocity  $u_\tau$ . All variables in (2.1) and (2.2) are non-dimensionalized by  $\delta$  and  $u_\tau$ .

The acceleration fields are computed by DNS of turbulent channel flow at  $Re_\tau = 180, 408$  and  $600$ . The Navier–Stokes equations were solved by using a spectral method: dealiased Fourier and Chebyshev expansions are used in the horizontal and the wall-normal directions, respectively. The time advancement was carried out by employing the Crank–Nicolson scheme for the viscous terms and a third-order Runge–Kutta scheme for the nonlinear terms (Lundbladh *et al.* 1999). Periodic boundary conditions are used in the horizontal directions. The computational domain size and the grid sizes in wall units are listed in table 1.

A  $128 \times 129 \times 128$  mesh has been used for DNS studies of turbulent channel flows at  $Re_\tau = 180$  and shown to be sufficient to resolve the velocity statistics (Moser, Kim & Mansour 1999). However, as the fluid acceleration is highly intermittent, it may need a finer resolution to estimate the acceleration statistics accurately. Figure 1 shows the flatness factors of the spanwise acceleration  $F_z$  obtained for three different resolutions at  $Re_\tau = 180$ . It is observed that  $F_z$  is slightly underestimated when using the usual  $128 \times 129 \times 128$  mesh. Therefore, we used a  $192 \times 129 \times 192$  mesh for  $Re_\tau = 180$  and the mesh sizes for higher  $Re_\tau$  are determined to keep the same resolution in the wall units.

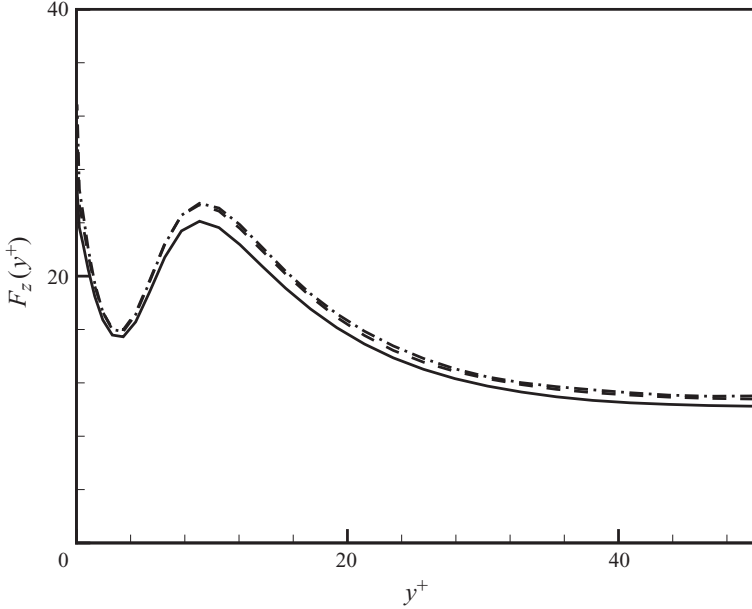


FIGURE 1. The flatness factors of the spanwise acceleration for different resolutions at  $Re_\tau = 180$ . Solid line,  $128 \times 129 \times 128$ ; dashed line,  $192 \times 129 \times 192$ ; dash-dot line,  $256 \times 129 \times 258$ .

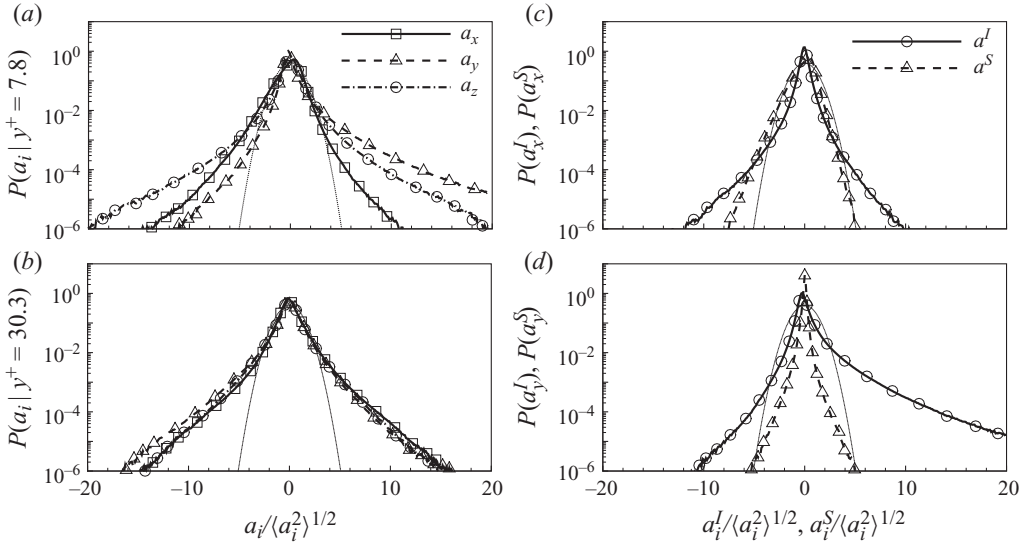


FIGURE 2. Probability density functions of acceleration fluctuations in (a) viscous ( $y^+ = 7.8$ ) and (b) buffer layers ( $y^+ = 30.3$ ) at  $Re_\tau = 180$ . (c) and (d) are, respectively, the p.d.f. of the streamwise and wall-normal components of  $a^I$  and  $a^S$  at  $y^+ = 7.8$ . All p.d.f.s are normalized by standard deviations of the total accelerations,  $\langle a_i^2 \rangle^{1/2}$ . The dotted lines in (a)–(d) are the Gaussian distribution.

### 3. Acceleration statistics

#### 3.1. Probability distribution of acceleration

The probability density functions (p.d.f.) for acceleration fluctuations at different wall-normal distances for  $Re_\tau = 180$  are shown in figure 2(a, b). Each p.d.f. is evaluated with

about  $3.7 \times 10^8$  samples and normalized by the root-mean-square (r.m.s.) accelerations  $\langle a_i'^2 \rangle^{1/2}$ , where  $\langle \cdot \rangle$  denotes ensemble average. Hereafter, we use lower-case characters to indicate fluctuating components and upper-case characters for mean components, unless mentioned otherwise. Consistently with the previous experiments (Voth *et al.* 2002) and DNS (Vedula & Yeung 1999) in isotropic turbulence, the acceleration p.d.f.s show long tails compared to that of the Gaussian distribution. Near the wall, the p.d.f.s of  $a_x$  and  $a_y$  are strongly skewed. The p.d.f. of  $a_x$  at  $y^+ = 7.8$  is negatively skewed whereas that of  $a_y$  is positively skewed, in which the superscript  $+$  denotes a variable normalized by the inner variables ( $u_\tau$  and  $\nu$ ). The p.d.f.s become nearly symmetric at  $y^+ = 30.3$ . As expected, the p.d.f. of  $a_z$  is always symmetric regardless of the distance from the wall.

It is interesting to observe that the acceleration p.d.f.s in the viscous sublayer (figure 2a) show wider tails than those in the buffer layer (figure 2b). In the p.d.f. of  $a_y$  at  $y^+ = 7.8$ , extreme events, 50 times larger than the r.m.s. value, are observed with the probability of  $10^{-7}$ . The maximum flatness factor of  $a_y$  is about 280 at  $y^+ \simeq 5$  (figure 4c), which is much larger than the flatness factor observed in isotropic turbulence; about 90 for  $Re_\lambda = 690$  (La Porta *et al.* 2001; Voth *et al.* 2002).

Acceleration in isotropic turbulence is dominated by the potential contribution  $\mathbf{a}^I$  (Monin & Yaglom 1975; Vedula & Yeung 1999; Tsinober *et al.* 2001). On the contrary, in turbulent channel flow, it is expected that the viscous contribution  $\mathbf{a}^S$  is dominant at least in the viscous sublayer. Figure 2(c, d) shows the p.d.f.s of  $(a_x^I, a_x^S)$  and  $(a_y^I, a_y^S)$  at  $y^+ = 7.8$ , respectively. All p.d.f.s are normalized by the r.m.s. value of the total acceleration  $\langle a_i'^2 \rangle^{1/2}$ . Interestingly, even in the wall region, accelerations larger than 10 r.m.s. values are mainly associated with  $\mathbf{a}^I$ . In the core of the p.d.f. ( $|a^S|/\langle a^2 \rangle^{1/2} \leq 5$ ), the probability density of  $a_x^S$  is greater than  $a_x^I$ . However,  $a_y$  is determined mostly by  $a_y^I$  except in a very narrow region near the origin. It is somewhat surprising to observe that  $\mathbf{a}^I$  is dominant even in the viscous sublayer.

### 3.2. One-point statistics

The mean streamwise  $A_x$  and wall-normal accelerations  $A_y$  normalized by the inner variables for three different  $Re_\tau$  are shown in figure 3(a, b). The negative peak of  $A_x$  appears at  $y^+ \simeq 7$  regardless of  $Re_\tau$ . From the mean momentum equation for the streamwise velocity,

$$A_x = -1 + \frac{1}{Re_\tau} \frac{d^2 U}{dy^2}, \quad (3.1)$$

it is obvious that the negative peak of  $A_x$  is due to the viscous contribution. Since the mean wall-normal velocity is zero,  $A_y$  is solely determined by the irrotational acceleration. The peak  $A_y$  is observed at  $y^+ \simeq 20$ . Moser *et al.* (1999) have shown that the Reynolds stresses scaled by the inner variables are insensitive to the Reynolds number once  $Re_\tau \geq 400$ . Because  $A_x$  and  $A_y$  are, respectively, wall-normal gradients of  $\langle uv \rangle$  and  $\langle v^2 \rangle$ ,  $A_x$  and  $A_y$  at  $Re_\tau = 408$  are almost indistinguishable from those at  $Re_\tau = 600$  for  $y^+ \leq 100$ .

Figure 3(c, d) shows the r.m.s. values of  $a_x$  and  $a_y$  scaled by the inner variables. We compared the present results with the experimental data in Gerashchenko *et al.* (2008) for the Reynolds number based on  $u_\tau$  and the boundary layer thickness  $Re_\delta = 470$ . Because they measured acceleration of finite-sized inertial particles, it is difficult to compare the results directly. However, considering the differences in physical parameters, the qualitative agreement in the r.m.s. acceleration profiles is quite satisfactory.

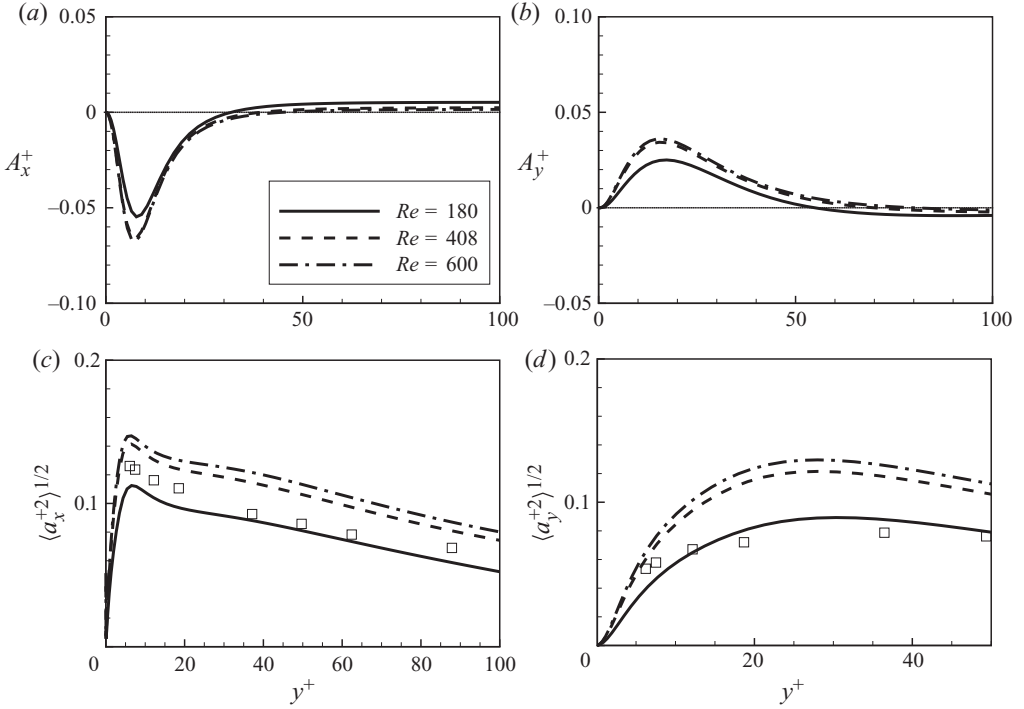


FIGURE 3. Mean and r.m.s. accelerations normalized by  $u_\tau$  and  $\delta$ : (a) mean streamwise acceleration, (b) mean wall-normal acceleration, (c) r.m.s. streamwise acceleration and (d) r.m.s. wall-normal acceleration in wall coordinates.  $\square$ , experiments by Gerashchenko *et al.* (2008).

In figure 3(c, d), it is shown that the location of the peak  $\langle a_x^2 \rangle^{1/2}$  coincides with that of  $A_x$ . Unlike the mean values, the r.m.s. accelerations are shown to be increasing functions of  $Re_\tau$ . However, in the viscous sublayer  $y^+ \leq 7$ , both the streamwise and wall-normal accelerations are scaled well with the inner variables when  $Re_\tau \geq 408$ . Except near the location of the peak  $A_i$ , the mean accelerations are very small compared to their r.m.s. values. At the locations of peak  $A_x$  and  $A_y$ ,  $\langle a_x^2 \rangle^{1/2}/A_x > 2$  and  $\langle a_y^2 \rangle^{1/2}/A_y > 3$ , in the range of  $Re_\tau$  in the present simulations.

To investigate the behaviour in detail, we compute the r.m.s. values of total, irrotational and solenoidal accelerations separately for  $Re_\tau = 180$  (figure 4a). The peaks of  $a_y$  and  $a_z$  appear in the buffer layer, where the distinguishing turbulent coherent structure of near-wall turbulence, the streamwise vortex, is observed. On the other hand,  $a_x$  reaches its maximum very near the wall,  $y^+ \simeq 7$ . It is shown that  $a_x^S$  is responsible for the peak  $a_x$  near the wall. The contribution of  $a_x^S$  to  $a_x$  is greater than that of  $a_x^I$  until  $y^+ \simeq 20$  and the relative magnitude  $|a_x^S|/|a_x^I|$  decreases slowly outwards. It is also found that  $\langle a_y^{S2} \rangle$  and  $\langle a_z^{S2} \rangle$  are negligibly small compared to their irrotational counterparts.

The skewness factor of  $a_y$  ( $S_y$ ) is shown in figure 4(b). It is shown that  $S_y$  reaches its positive peak at  $y^+ \simeq 7.8$  and crosses zero at  $y^+ \simeq 20$ . This behaviour may suggest the relation between acceleration and a turbulent coherent structure. Kim, Moin & Moser (1987) have shown that the averaged centre of the streamwise vortices is located at  $y^+ \simeq 20$  with the averaged radius of  $r^+ \simeq 15$ . Because accelerations tend to be directed towards the core of a vortex (Lee *et al.* 2004; Lee & Lee 2005), the

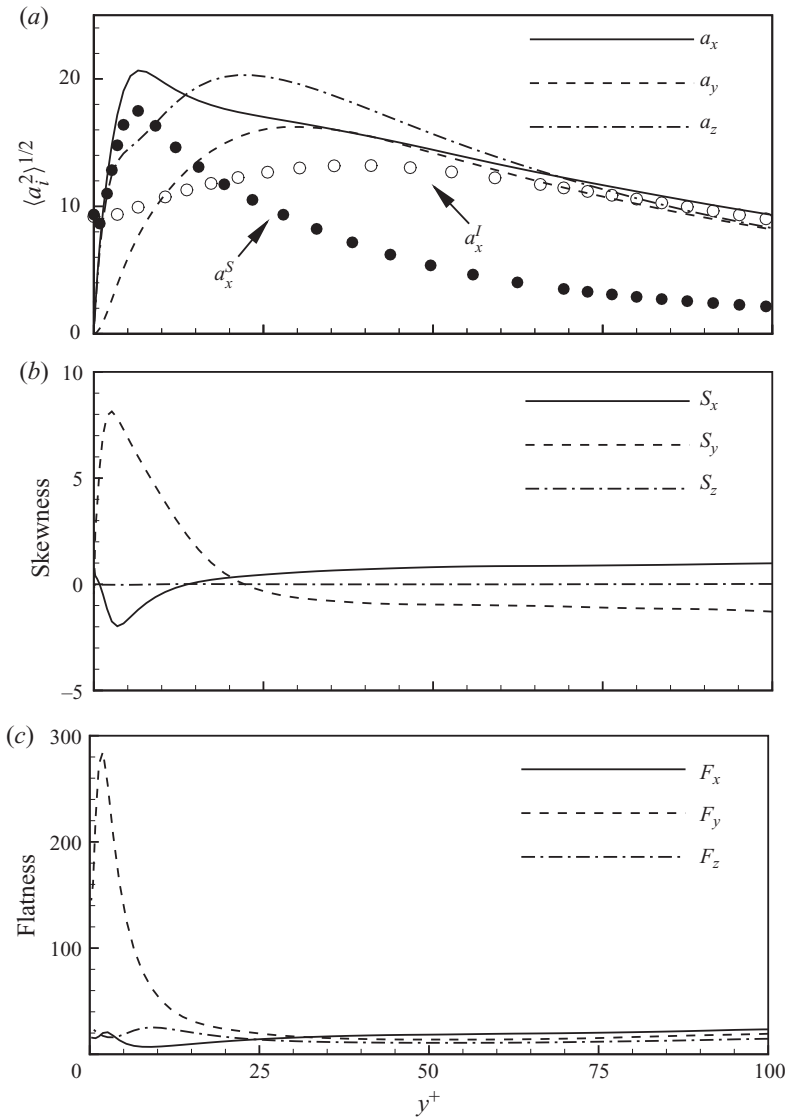


FIGURE 4. Acceleration statistics for  $Re_\tau = 180$  as a function of  $y^+$ : (a) r.m.s. values of total, irrotational and solenoidal accelerations scaled by the wall units, (b) skewness and (c) flatness factors.

upward acceleration appearing below the vortex contributes to the positive  $S_y$  when  $y^+ < 20$ . Likewise, the negative  $S_y$  may be related to the downward acceleration above the streamwise vortex.

The flatness factors are presented in figure 4(c). The flatness factors are evaluated with about  $4.7 \times 10^9$  samples to ensure convergence. The flatness factor of  $a_y$  ( $F_y$ ) shows an extremely large peak in the viscous sublayer, larger than 280. The large  $F_y$  in the wall-region is due to the long positive tail of the  $a_y$  p.d.f. shown in figure 1(a). A close investigation of flow fields reveals that the streamwise vortices are extended from the buffer layer to the viscous sublayer causing very large upward acceleration in the viscous sublayer, which contributes to the large  $F_y$  near the wall.

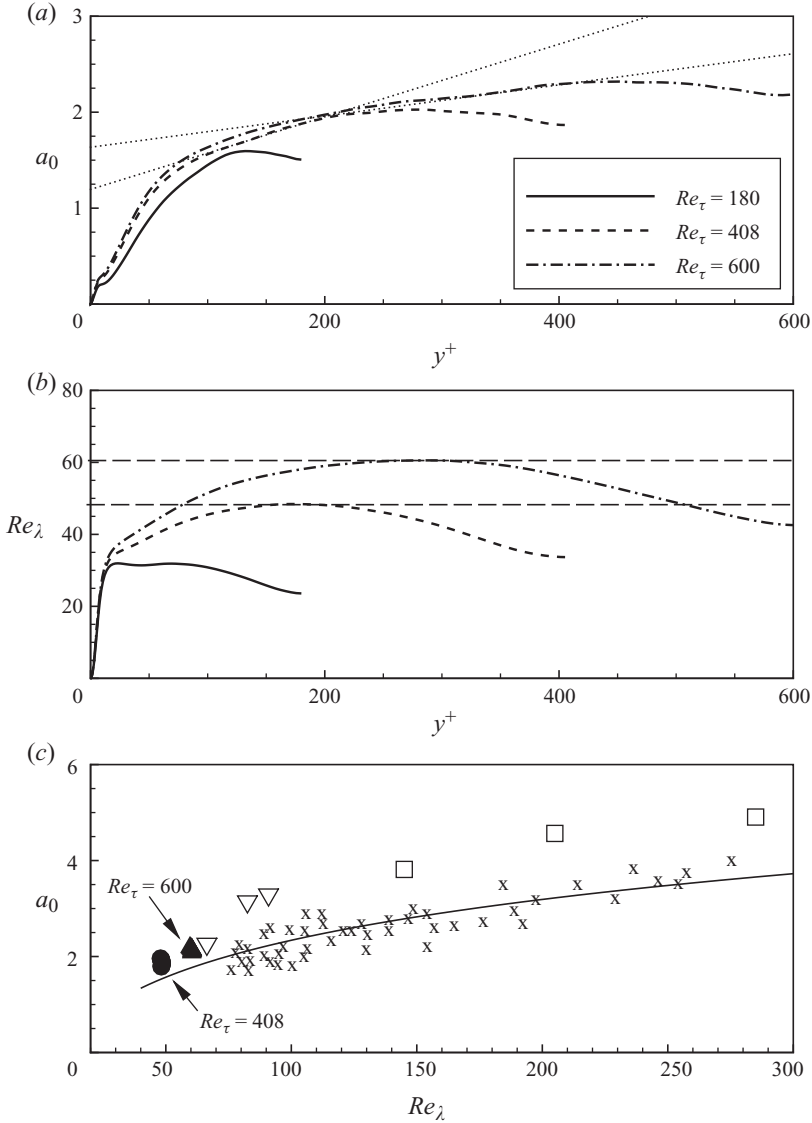


FIGURE 5. (a) The acceleration variance normalized by the Heisenberg–Yaglom scaling  $a_0$  and (b) Taylor-scale Reynolds number  $Re_\lambda$  as function of  $y^+$  for three different  $Re_\tau$ . (c)  $a_0$  as a function of  $Re_\lambda$ : ●,  $Re_\tau = 408$ ; ▲,  $Re_\tau = 600$ ; ×, Vedula & Yeung (1999); ▽, Lüthi *et al.* (2005); □, Voth *et al.* (2002); —, Yeung *et al.* (2006).

### 3.3. Reynolds number dependence of acceleration statistics

The acceleration variance normalized by the Heisenberg–Yaglom scaling (Monin & Yaglom 1975) is defined as

$$a_0 \equiv \frac{1}{3} \frac{\langle \mathbf{a} \cdot \mathbf{a} \rangle}{\epsilon^{3/2} \nu^{-1/2}}, \tag{3.2}$$

in which  $\epsilon$  is the dissipation rate. Figure 5(a) shows  $a_0$  as a function of  $y^+$ . It is shown that  $a_0$  is a non-decreasing function of  $Re_\tau$  near the wall. In the viscous sublayer ( $y^+ \leq 10$ ),  $a_0$  collapses well for the two-higher  $Re_\tau$  due to the universal



characteristic of the solenoidal acceleration (Vedula & Yeung 1999). Interestingly,  $a_0$  is a linear function of  $y^+$  in the log layer for  $Re_\tau \geq 408$ . The fitting curves are  $a_0 = 1.20 + 3.78 \times 10^{-3}y^+$  and  $a_0 = 1.64 + 1.63 \times 10^{-3}y^+$  for  $Re_\tau = 408$  and 600, respectively. The slope of the fitting curve decreases significantly as  $Re_\tau$  increases.

To compare  $a_0$  with the previous results for isotropic turbulence, we estimate the Taylor-scale Reynolds number as a function of  $y^+$ . The Taylor-scale Reynolds number in inhomogeneous turbulence is estimated by  $Re_\lambda = (20k^2/3\epsilon^+)^{1/2}$ , in which  $k^+$  is the turbulent kinetic energy scaled by the inner variables. Figure 5(b) shows  $Re_\lambda$  profiles for different  $Re_\tau$ . For  $Re_\tau = 180$ ,  $Re_\lambda$  shows a local peak in the buffer layer and slowly decreases afterwards. On the other hand, plateaus are observed in the log layer at  $Re_\tau = 408$  and 600.

In figure 5(c), we compare  $a_0$  at the plateaus for  $Re_\tau = 408$  and 600 with the previous numerical (Vedula & Yeung 1999) and experimental results (Voth *et al.* 2002; Lüthi, Tsinober & Kinzelbach 2005). In the experiments by Voth *et al.* (2002),  $a_0$  for axial and lateral accelerations do not coincide at low  $Re_\lambda$ . In figure 5(c), we show the average of the axial and lateral  $a_0$ . The results by Vedula & Yeung (1999) is obtained from Gylfason, Ayyalasomayajula & Warhaft (2004). According to the classical Kolmogorov scaling,  $a_0$  may show a universal behaviour at sufficiently high  $a_0$ . However, it has been observed that  $a_0$  shows a Reynolds number dependence even at  $Re_\lambda \sim 10^4$  (Gulitski *et al.* 2007). Yeung *et al.* (2006) provided an empirical fitting curve for  $a_0$ ,

$$a_0 = \frac{1.3}{Re_\lambda^{0.22}} + 0.88Re_\lambda^{0.66} \ln\left(\frac{Re_\lambda}{20}\right). \quad (3.3)$$

It is interesting to observe that  $a_0$  estimated at the plateau of  $Re_\lambda$  shows a good agreement with the previous results in isotropic turbulence. From figure 5(a, b), it seems that, at higher  $Re_\tau$ , there will be a region in which both  $Re_\lambda$  and  $a_0$  reach plateaus and  $a_0$  may be estimated from the data in isotropic turbulence.

Figure 6(a, b) shows the  $a_y$  skewness and flatness factors for three different  $Re_\tau$ . In isotropic turbulence, it has been shown that the acceleration flatness factor is an increasing function of  $Re_\lambda$ . In the DNS study of Yeung *et al.* (2006), the acceleration flatness factor increases from 7 at  $Re_\lambda = 7$  to 107 at  $Re_\lambda = 680$ . On the contrary, it is found that, near the wall ( $10 < y^+ < 100$ ), the acceleration skewness and flatness factors are relatively less sensitive to the Reynolds numbers except in the viscous sublayer. In the viscous sublayer, both the skewness and flatness factors are increasing functions of  $Re_\tau$ .

## 4. Behaviour of accelerations around vortices

### 4.1. Alignment of accelerations with vortices

In a vortex, pressure has its local minimum on the axis of the swirling motion because the centrifugal force is balanced by pressure gradient (Jeong *et al.* 1997). As the major contribution to acceleration comes from the pressure gradient, it is natural that acceleration is directed towards the core of a vortex. A strong correlation between a streamwise vortex and pressure gradient was first noticed by Kim (1989) in turbulent channel flow, in which he found elongated structures of  $\partial p/\partial y$  and  $\partial p/\partial z$  in the streamwise direction and a strong correlation between  $\partial p/\partial z$  and streamwise vorticity at the wall. By investigating the acceleration–enstrophy cross-correlation function in isotropic turbulence, Yeung & Pope (1989) suggested that the fluid particles are accelerated towards regions of high vorticity. La Porta *et al.*

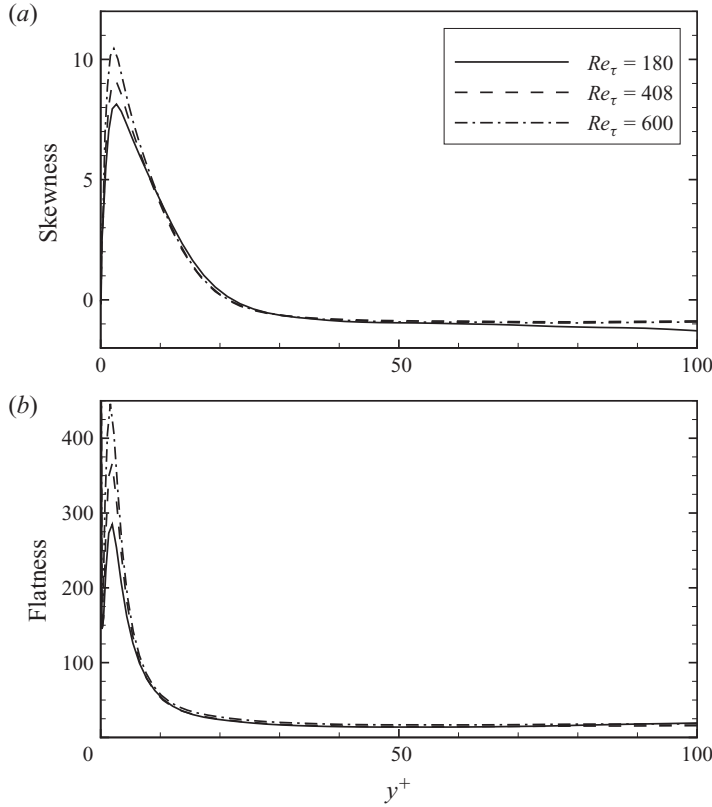


FIGURE 6. Acceleration (a) skewness and (b) flatness factors for three different  $Re_\tau$ .

(2001) observed that intermittent accelerations are associated with spiral motion of fluid particles around vortices. Based on the results, Mordant *et al.* (2002) suggested that the slowly decaying Lagrangian correlation of acceleration magnitude is closely associated with the intermittency. Later, Lee *et al.* (2004) confirmed that the long-time correlation of acceleration magnitude is due to the centripetal acceleration of fluid particles trapped in a vortex.

To investigate the behaviour of acceleration near vortices quantitatively, we investigate the angles between accelerations and the gradient of squared vorticity known as the enstrophy  $\boldsymbol{\omega} \cdot \boldsymbol{\omega}$ , in which  $\boldsymbol{\omega}$  is the fluctuating component of vorticity. We define the angle between  $\boldsymbol{a}$  and  $\nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})$  as

$$\alpha = \cos^{-1} \left( \frac{\boldsymbol{a} \cdot \nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})}{|\boldsymbol{a}| |\nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})|} \right). \tag{4.1}$$

A region of high  $\boldsymbol{\omega} \cdot \boldsymbol{\omega}$  is strongly correlated with solid-body rotation of fluid and, hence, vortex tubes (Ruetsch & Maxey 1992; Blackburn, Mansour & Cantwell 1996). Since, near a vortex tube,  $\boldsymbol{\omega} \cdot \boldsymbol{\omega}$  has a local maximum along the axis of the vortex and decreases radially,  $\nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})$  is perpendicular to the vortex axis.

Figure 7(a, b) illustrates the p.d.f.s of  $\alpha$ ,  $P(\alpha)$ . In the viscous sublayer,  $P(\alpha)$ ,  $P(\alpha')$  and  $P(\alpha^S)$  show peaks around  $90^\circ$ . Because the direction of irrotational acceleration near a vortex converges into the core of the vortex, the peak of  $P(\alpha')$  at  $90^\circ$  is not an expected result. Even in the buffer layer (figure 7b),  $P(\alpha')$  does not show a

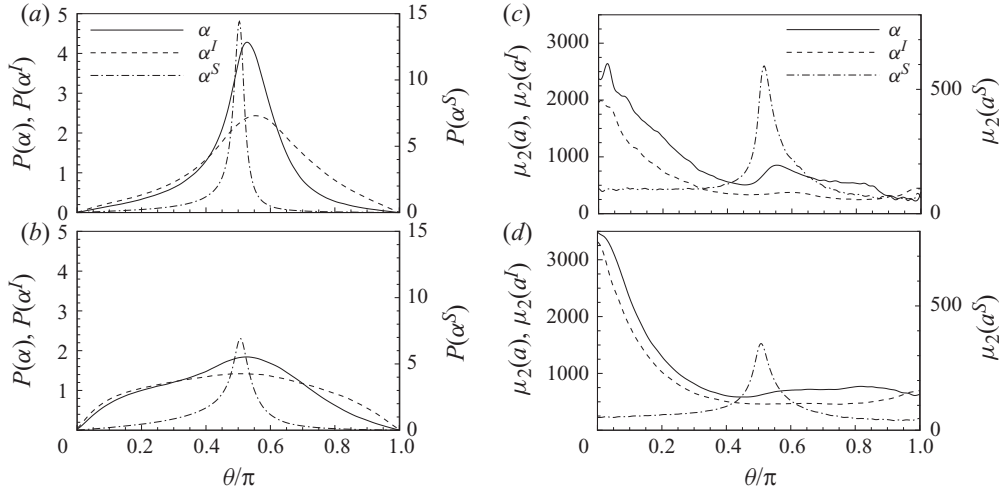


FIGURE 7. (a, b) Probability density functions of angles between  $\nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})$  and  $\boldsymbol{a}$  ( $\alpha$ ),  $\boldsymbol{a}^I$  ( $\alpha^I$ ) and  $\boldsymbol{a}^S$  ( $\alpha^S$ ) at  $y^+ = 7.8$  and  $y^+ = 19.8$ , respectively. (c, d) The conditional second moments of acceleration,  $\mu_2(a|\alpha)$ ,  $\mu_2(a^I|\alpha^I)$  and  $\mu_2(a^S|\alpha^S)$  at  $y^+ = 7.8$  and at  $y^+ = 19.8$ , respectively. The results are obtained at  $Re_\tau = 180$ .

preferential alignment with the enstrophy gradient.  $P(\alpha)$  has a weak peak at  $\alpha = 90^\circ$  in the buffer layer.  $\boldsymbol{a}^S$  is perpendicular to the enstrophy gradient regardless of the distances from the wall. The weak peak of  $P(\alpha)$  observed in the buffer layer seems to be associated with the non-negligible contribution of  $a_x^S$  to  $\langle a_x^2 \rangle^{1/2}$  (figure 4a).

The intermittent accelerations, more than 20 times larger than r.m.s. acceleration, are typically observed around vortical structures (Lee *et al.* 2004; Lee & Lee 2005). To consider the intermittent events, we computed the conditional acceleration variances,  $\mu_2(a|\alpha) = \langle (\boldsymbol{a} \cdot \boldsymbol{a}) | \alpha \rangle$ . Figure 7(c, d) shows that the large-magnitude events of  $\boldsymbol{a}$  and  $\boldsymbol{a}^I$  are always aligned with the enstrophy gradient, implying that a major source of the intermittently large accelerations is centripetal accelerations near vortices. On the other hand,  $\mu_2(a^S|\alpha^S)$  shows a peak at  $\alpha^S = 90^\circ$  regardless of the wall distances. This behaviour of the solenoidal acceleration is explained in §4.2. It is also observed that the acceleration events of magnitudes similar to the mean accelerations show no directional preference to the enstrophy gradient.

#### 4.2. Acceleration fields near the wall

Figure 8(a–c) shows acceleration fields near a streamwise vortex rotating clockwise. The vortex structure is identified by the  $\lambda_2$  method proposed by Jeong & Hussain (1995). It is shown that  $(a_y, a_z)$  are almost indistinguishable from  $(a_y^I, a_z^I)$ , while there are some differences in the  $x$ -components (contours) due to non-negligible contribution of  $a_x^S$  to  $a_x$ . In figure 8(a), large positive and negative  $a_x$  are observed above and below the vortex, respectively.  $\boldsymbol{a}^S$  rotates around the vortex in the counter-clockwise direction while the vortex rotates clockwise (figure 8c). Note that the magnitude of the reference vectors in figure 8(c) is different from figure 8(a, b), because the magnitudes of  $(a_y^S, a_z^S)$  are much smaller than the total and irrotational accelerations. From the Navier–Stokes equations, the relation between solenoidal acceleration and vorticity is given by

$$\nabla \times \boldsymbol{a}^S = \nu \nabla^2 \boldsymbol{\omega}. \quad (4.2)$$

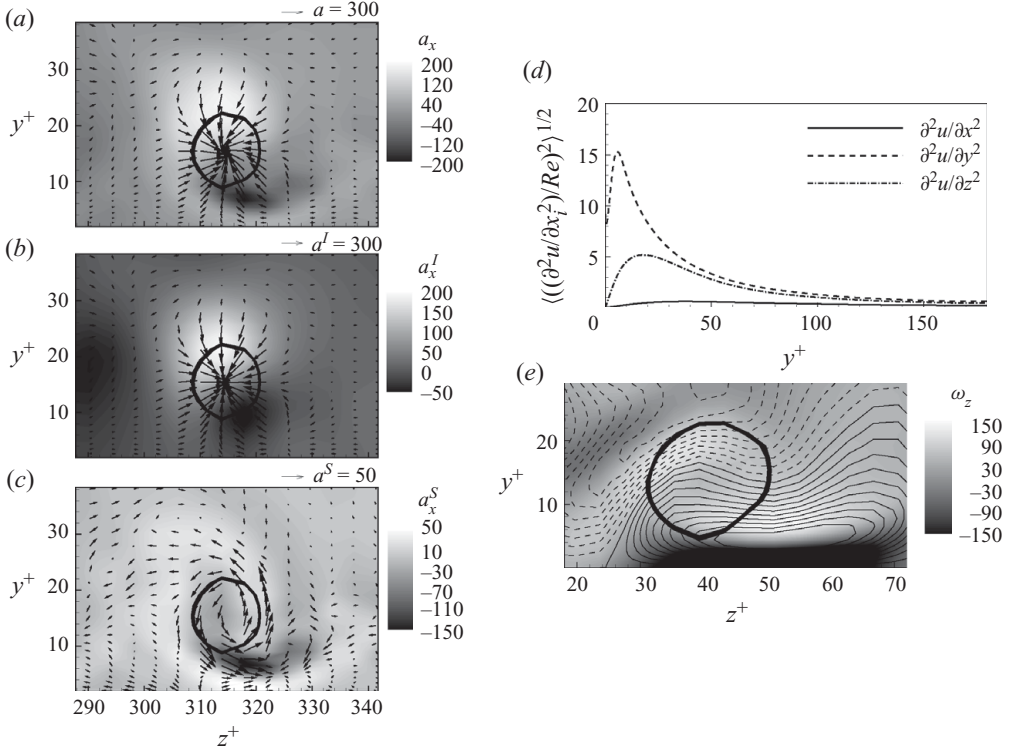


FIGURE 8. (a–c) Display of  $\mathbf{a}$ ,  $\mathbf{a}^I$  and  $\mathbf{a}^S$  fields around a vortex structure. Solid line denotes the boundary of a streamwise vortex rotating clockwise. Shading shows the streamwise components of each acceleration. (d) Root-mean-square values of  $v^2 u / \partial x_i^2$ . (e) High- and low-speed streaks and  $\omega_z$  field around a streamwise vortex in buffer layer. Grey shading denotes  $\omega_z$ . The solid and dashed lines represent positive and negative  $u$ , respectively.

In a simple one-dimensional vortex tube, the vorticity has a local maximum along the axis and decreases radially and the sign of the right-hand side of (4.2) is always opposite to the sign of  $\boldsymbol{\omega}$ . Hence, it can be deduced that  $a_y^S$  and  $a_z^S$  always counter-rotate with respect to the motion of the streamwise vortex. Because of the counter-rotating nature of  $\mathbf{a}^S$ , the p.d.f.s of the angle between  $\mathbf{a}^S$  and  $\nabla(\boldsymbol{\omega} \cdot \boldsymbol{\omega})$  show a peak at  $\alpha^S = 90^\circ$  (figure 7a, b).

It is seen in figure 8 that the positive  $a_x$  above a vortex resembles  $a_x^I$  while the negative one below the vortex is determined by  $a_x^S$ . The positive and negative  $a_x^I$ , displayed in figure 8(b), are induced by the streamwise vortex which is inclined in the wall-normal direction (Lee *et al.* 2004). Typically, the large negative  $a_x^S$  occurs below the vortex, forming a sheet-like structure (figure 8c). It seems that the sheet-like structure contributes to the peak r.m.s.  $a_x^S$  found in the viscous sublayer (figure 3a). It is noteworthy to observe that the magnitude of  $a_x^S$  under the streamwise vortex is about 150 while  $A_x^S$  is about 10 in this region.  $a_x^S$  consists of three components,

$$a_x^S = v \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right). \quad (4.3)$$

Root-mean-square values of each component are presented in figure 8(d) showing that  $\partial^2 u / \partial y^2$  is a major source of  $a_x^S$  near the wall. Furthermore,  $|\partial v / \partial x| \ll |\partial u / \partial y|$ , near the streamwise vortex. Hence, near the vortex,  $a_x^S$  may be approximated by

$\partial\omega_z/\partial y$ . Jeong *et al.* (1997) have shown that high- and low-shear regions result from the streamwise vortex near the wall and  $\omega_z$  changes its sign in the high-shear region. Figure 8(e) shows the distribution of  $\omega_z$  near the streamwise vortex rotating clockwise. Below the vortex, there is a region in which  $\omega_z$  changes rapidly from positive to negative and the region coincides with the location of a high-speed streak. Comparing figures 8(c) and 8(e), it seems that the sheet-like structure of  $a_x^S$  in the viscous sublayer is induced by the large  $\partial\omega_z/\partial y$  below the streamwise vortex. The peak of r.m.s.  $a_x^S$  in the viscous sublayer is a consequence of the shear zone induced by the streamwise vortex.

## 5. Conclusions

To investigate the characteristics of acceleration in near-wall turbulent flows, DNS of turbulent channel flow were performed at  $Re_\tau = 180, 408$  and  $600$ . The fluid-particle acceleration is evaluated by summing the pressure-gradient (irrotational acceleration) and viscous forces (solenoidal acceleration).

It is shown that the acceleration field becomes more intermittent near the wall. Particularly, the p.d.f. of the wall-normal acceleration in the viscous sublayer shows extreme events,  $a_y \simeq 50\langle a_y^2 \rangle^{1/2}$ , with the probability of  $10^{-7}$ . In the viscous layer, the  $a_y$  flatness factor becomes  $O(100)$ . It is shown that such an intermittent event is governed by the irrotational acceleration, which is linked to vortical structures. The acceleration variances conditioned on the angle between the acceleration and the enstrophy gradient indicate that a vortex structure is an important source of the acceleration intermittency even in the viscous sublayer. In the viscous sublayer, the unconditioned low-order statistics, such as mean and variances, of  $\mathbf{a}$  are largely due to  $\mathbf{a}^S$ , while intermittent events mainly come from the irrotational accelerations associated with streamwise vortices in the buffer layer. It is shown that, unlike the previous results in isotropic turbulence, the acceleration flatness and skewness factors are not sensitive to  $Re_\tau$  in the buffer and log layers ( $10 < y^+ < 100$ ).

In isotropic turbulence,  $\mathbf{a}^S$  is negligible compared to  $\mathbf{a}^I$ . However, in near-wall turbulence, the contribution from the streamwise solenoidal acceleration is non-negligible at least in the low-order statistics of  $a_x$ , while  $a_y$  and  $a_z$  are governed by the irrotational accelerations. Visualization of acceleration fields near a vortex shows that  $\mathbf{a}^I$  is directed towards the core of a vortex while  $\mathbf{a}^S$  counter-rotates with respect to the motion of the vortex. It is found that the streamwise vortex is responsible not only for the intermittent characteristics of irrotational acceleration but also for the large streamwise solenoidal accelerations observed in the viscous sublayer. Below a streamwise vortex, there is a high-shear region in which a steep gradient of the spanwise vorticity  $|\partial\omega_z/\partial y|$  is observed. The sheet-like structure of strong negative  $a_x^S$  is induced by the large  $|\partial\omega_z/\partial y|$ . The magnitude of  $a_x^S$  in this region is comparable to  $a_x^I$  in the viscous layer. It is interesting to observe that the location of the large negative  $a_x^S$  coincides with the location of a high-speed streak, indicating that the solenoidal acceleration acts to decelerate high-momentum fluid from the buffer layer to the viscous layer by the sweep events.

The Reynolds number behaviour of the acceleration Kolmogorov constant  $a_0$  is investigated. It is found that  $a_0$  is a linear function of  $y^+$  in the log layer when  $Re_\tau \geq 408$ . The slope of the fitting curve is a decreasing function of  $Re_\tau$ , implying that at sufficiently high  $Re_\tau$  there may be a region in the log layer in which  $a_0$  is a constant. The Taylor-scale Reynolds number reaches a plateau in the log layer for

$Re_\tau \leq 408$ . The value of  $a_0$  in the plateau is compared with the previous numerical and experimental data in isotropic turbulence.

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